



BETA^B
SIMULATION SOLUTIONS

**Ground breaking
Simulation Solutions**

physics on screen

Evaluation of Interior Noise Based on Loads from a Multi Body Simulation

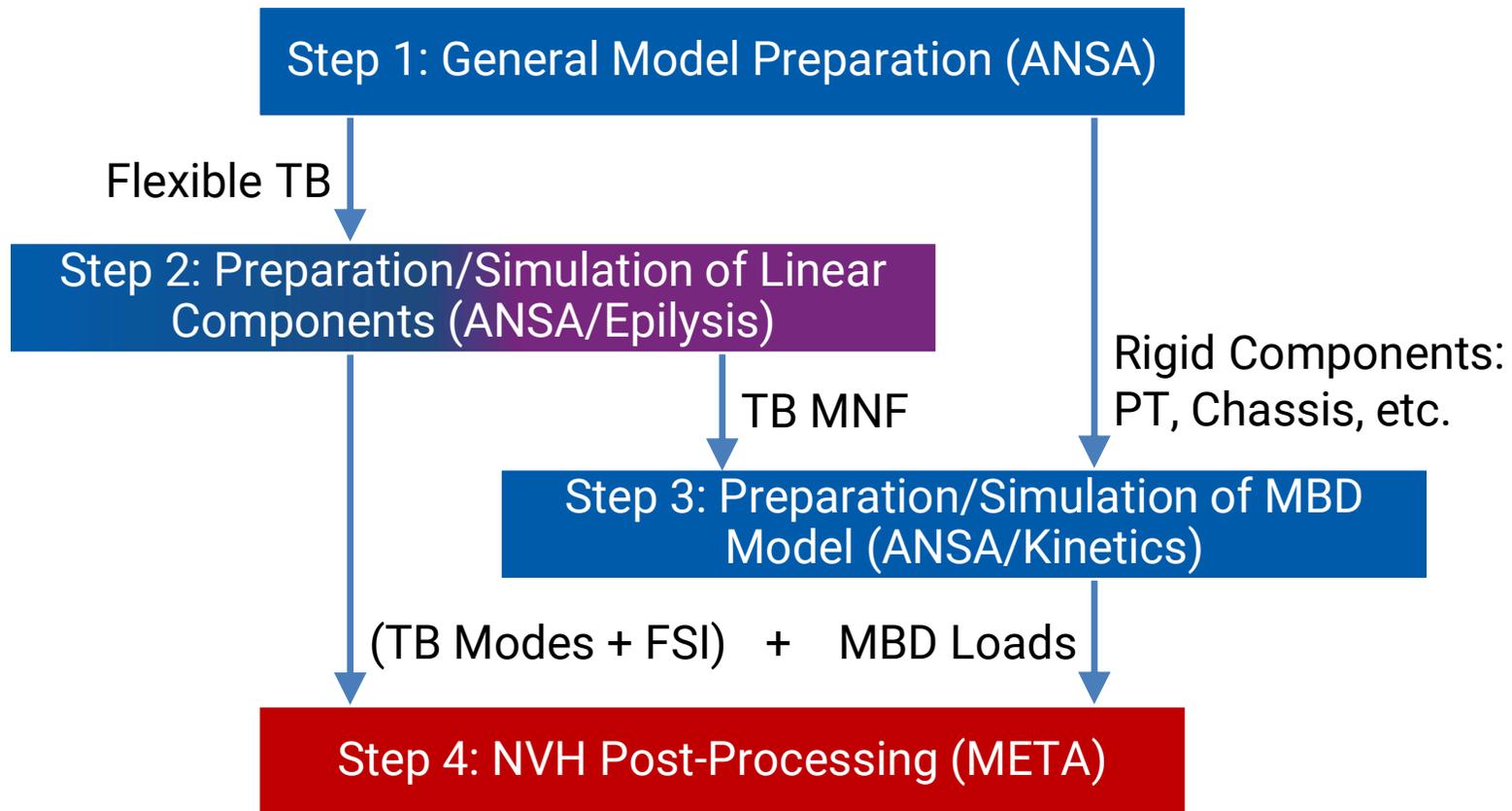
An Integrated BETA Suite Solution for Addressing Non-Linear
Mounts

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Background

- **Ride comfort and NVH performance** are important criteria for any car development process
- Classical NVH analysis uses linear FE models and linear calculation algorithms. However, important structural parts of a vehicle chassis have non-linear properties.
- MultiBody Dynamics (MBD) analysis is a method to calculate the dynamics of complex systems, like a car in driving conditions, including non-linear effects
- We are offering a process inside the BETA suit that combines MBD in ANSA Kinetics with linear NVH calculations on a Trimmed Body

Process

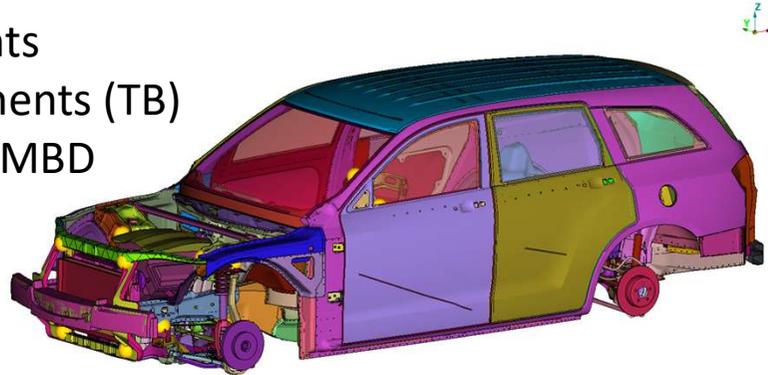


Detailed Process Description

Step 1: General Model Preparation

Structural Components

- Flexible Components (TB)
- Rigid Bodies for MBD



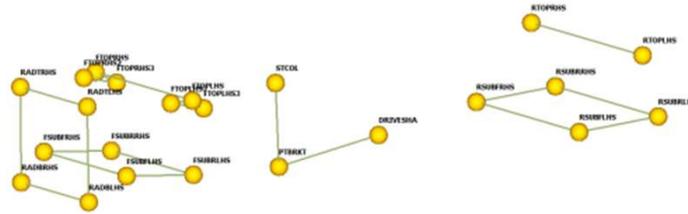
Acoustic Cavity with ANSA Cavity mesh tool



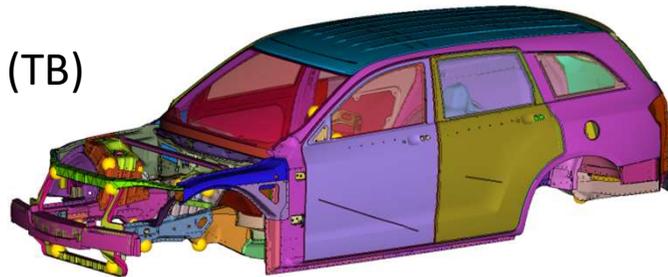
1. Provision of all required components (including acoustic cavity)
2. Preparation of TB for linear solver, e.g.
 - Connections
 - Trim masses
 - Properties/Materials
3. Preparation of Subframes, Powertrain, Engine, Radiator for MBD solver:
 - Masses, Inertias
4. Preparation of cavity mesh

Step 1: General Model Preparation

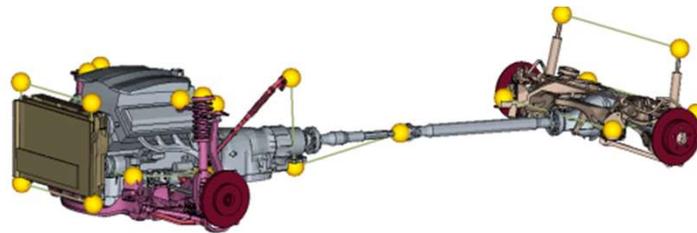
Interface points



Flexible Component (TB)
ANSA data base I

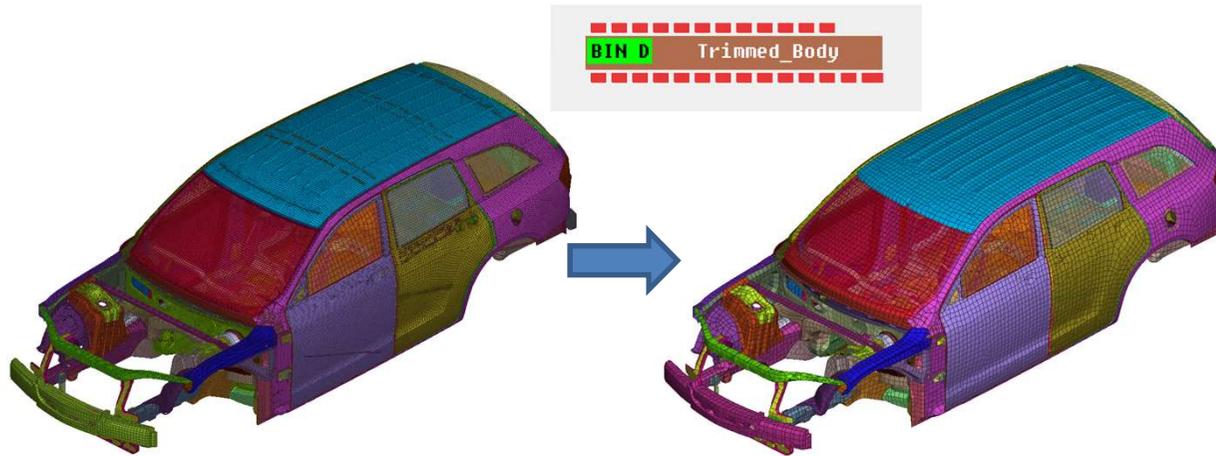


MBD Components
ANSA data base II



- Definition of interface points
 - Common position in linear model and MBD model
 - Field 10 names for NVH-Console and grid ID scheme for force table definition
- Divide ANSA data base into
 - TB for linear calculation
 - Rigid Body components for MBD

Step 2: NVH Model Preparation – Display Model



Features Full FE:

- 900K elements
- 750K grids

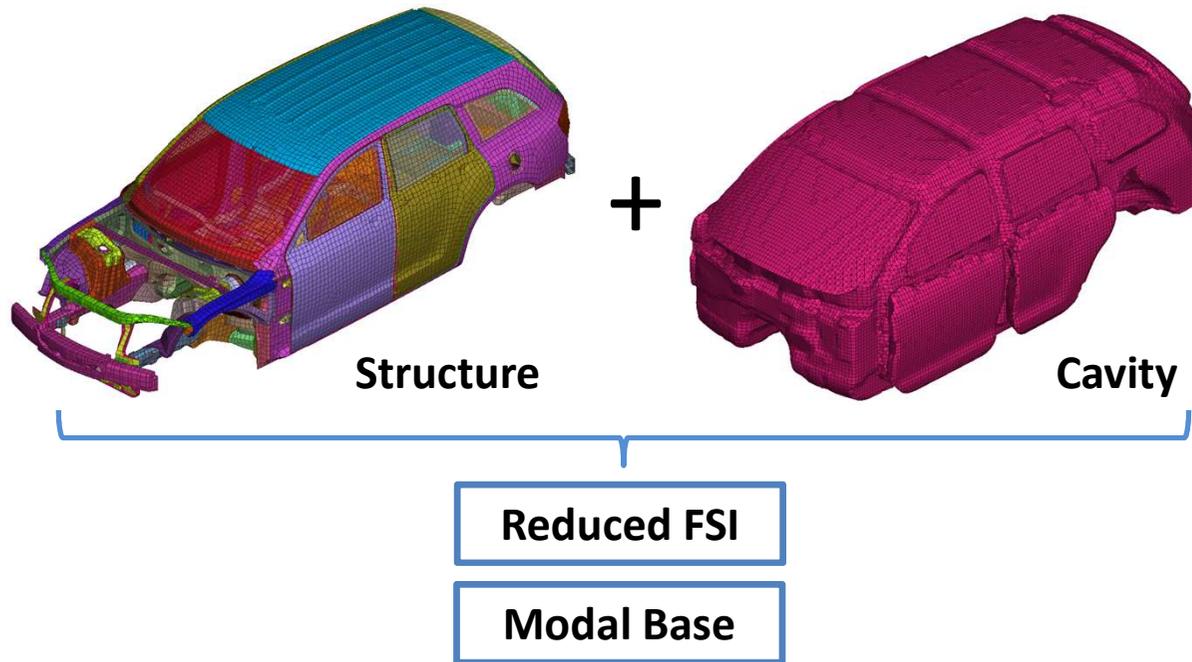
Features Display Model:

- 60K elements
- 60K grids (subset of Full FE grids)

- 
- Preparation of model from data base I in ANSA NVH-Console
 - NVH-C model consists of one component (TB with cavity)
 - NVH-C Step I: Creation of Display Model in NVH-C as model reduction technique

Display Model in NVH-C is Key Enabler for Manageable File Size of Modal Base

Step 2: Calculation of FSI and Modal Base

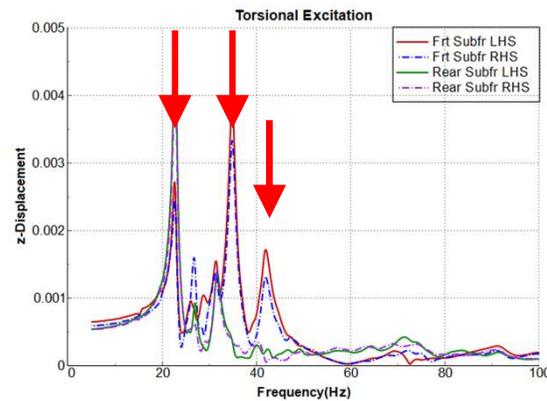
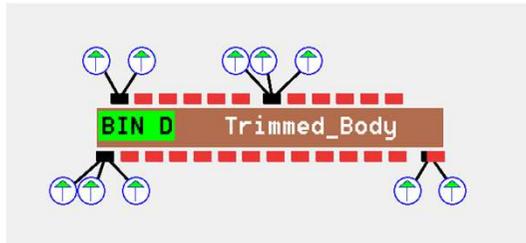
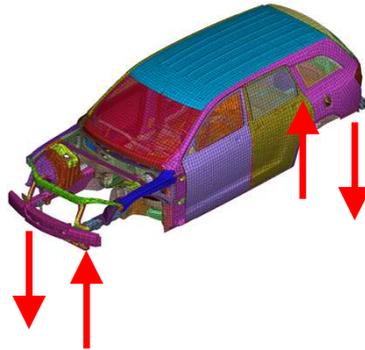
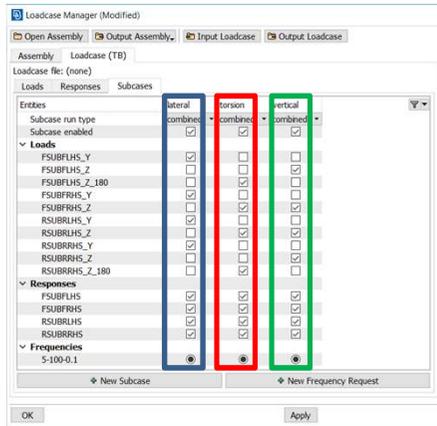


Epilysis/AMLS is Key Enabler for Efficient Calculation of Modal Base



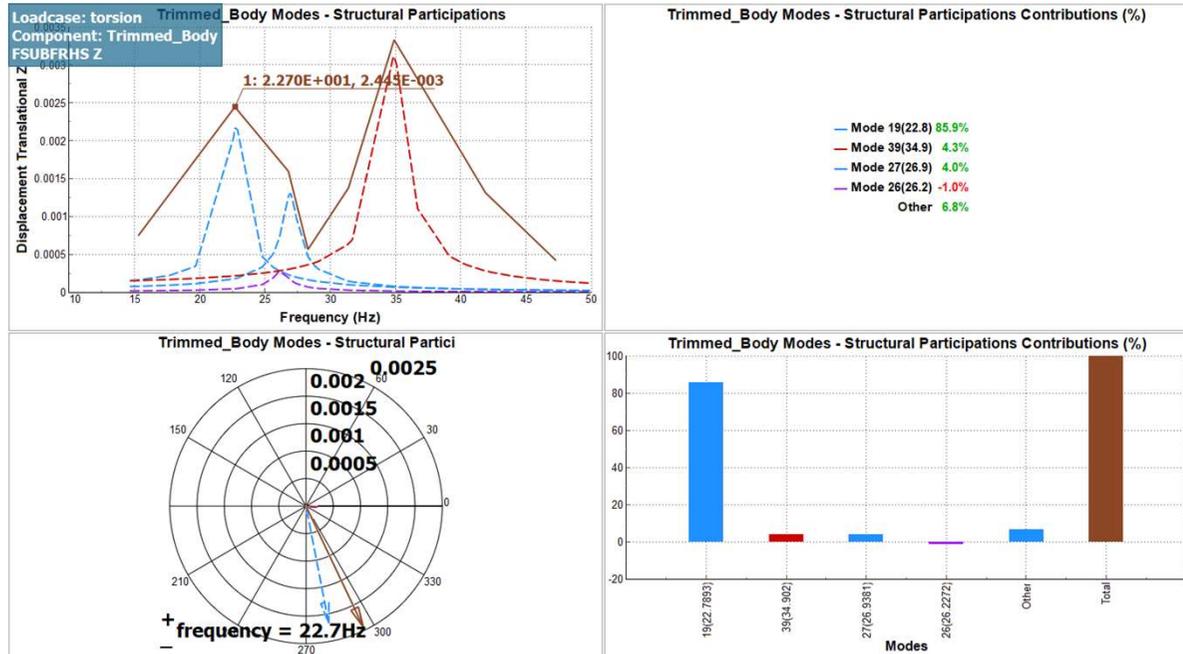
- NVH-C Step II: Creation of reduced rep. of Fluid Structure Interaction (FSI)
- NVH-C Step III: Preparation of SOL103 decks for calculation of modes:
 - Cut-off 250Hz
 - With Resvec
- NVH-C Step IV: Solving SOL103 decks with Epilysis/AMLS

Step 2: Global Modes Analysis



- NVH-Console: Easy set up of dynamic load cases to excite global
 - Lateral modes
 - Torsional modes
 - Vertical modes
- Response curves, calculated from modal base, show the resonance frequencies
- Example Torsional mode: 22.8Hz, 34.4Hz, 41.7Hz

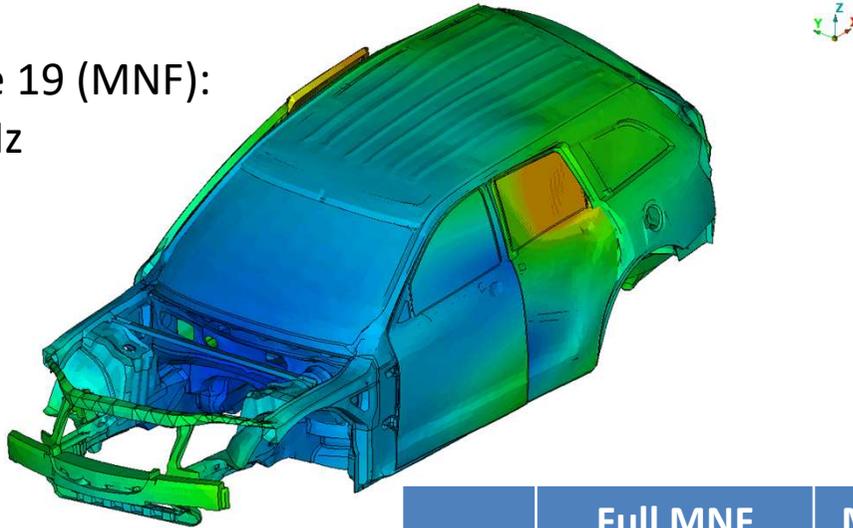
Step 2: Global Modes Analysis



- NVH-C: Easy set-up of modal participation calculation for dynamic responses
- NVH-C controls META via session files
- Example: Modal participation for torsional load case
- **Result:** Main contribution to torsional resonance at 22.8Hz: Mode 19

Step 2: Calculation of MNF File

Mode 19 (MNF):
22.8Hz



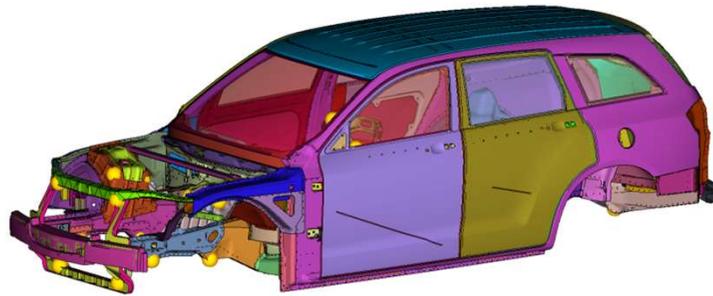
	Full MNF	MNF Display
Size	7.0 GB	600 MB

Using a Display Model is Key Enabler for Manageable File Size of MNF used in MBD Run

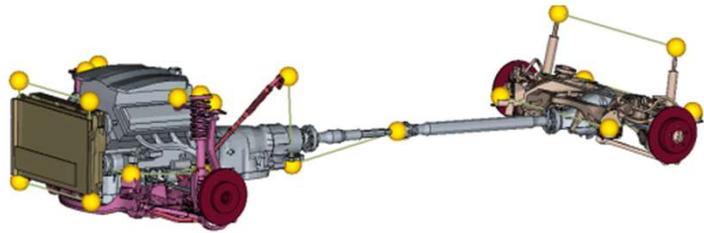
EPILYSIS
SOLVER

- Epilysis can calculate MNF file using (NVH) **Display Model** which leads to drastically reduced MNF file size (**new feature!**)
- MNF can be displayed/checked in META
- **Cut-off frequency** for MNF calculation taken from global mode analysis (50Hz)

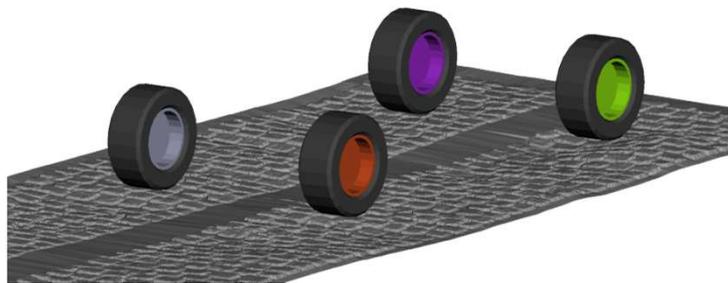
Step 3: Kinetics – Model Preparation



Trimmed Body
(Flex Body/Rigid Body)



Suspension, Powertrain,
Radiator (Rigid Bodies)



Tire-Road Interaction



Flex Body (Trimmed Body) and Rigid Bodies
(Suspension, Powertrain, Radiator)

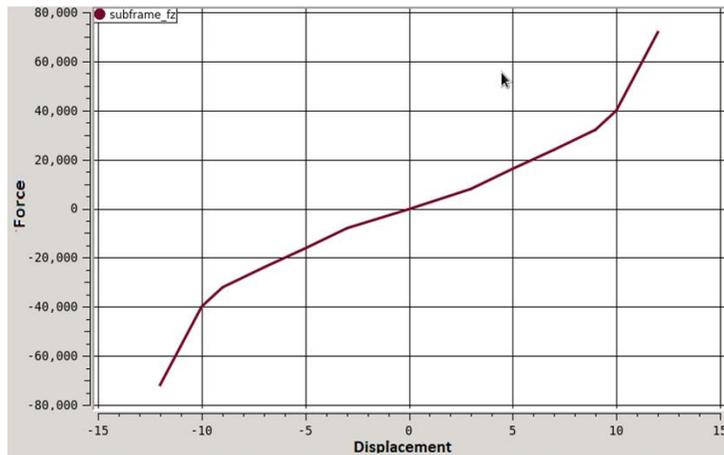
- Require the definition of bodies, joints, bushings

Tire-Road Interaction

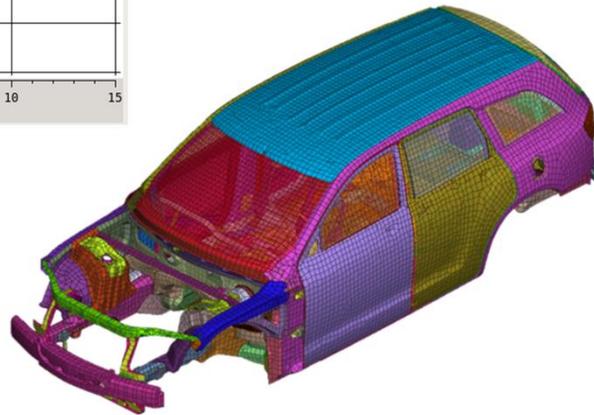
- Requires tire and road entity definitions

Step 3: Kinetics – Model Preparation

Frt Subfr, z-dir: F(s) curve

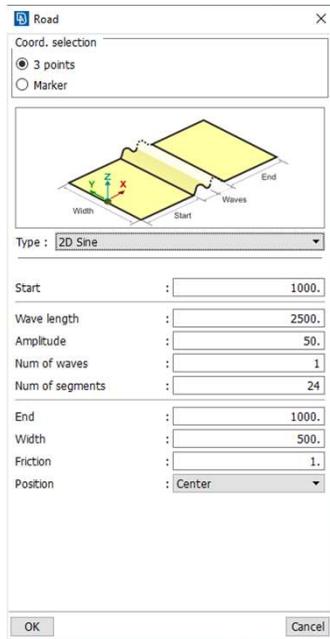


NVH Display Model used for Flex Body in MBD

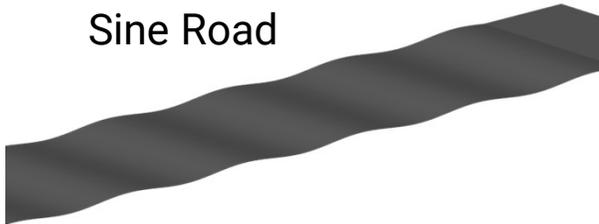


- Non-Linear stiffness curves of bushings via:
 - Tables
 - User expressions
 - Python scripts
- Trimmed Body variants:
 - Rigid
 - Flex Body/Full MNF
 - Flex Body/Display MNF
- MNF (Full/Display) can be created directly in ANSA Kinetics

Step 3: Kinetics – Tire-Road Interaction



Sine Road



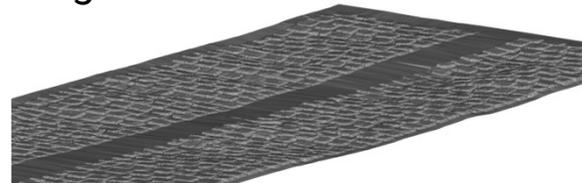
PAC2002

- Tire formulation embedded in ANSA Kinetics
- For road frequencies < 8Hz

FTire

- Advanced tire formulation including effects of tire belt
- For road frequencies < 250Hz

Belgian Block

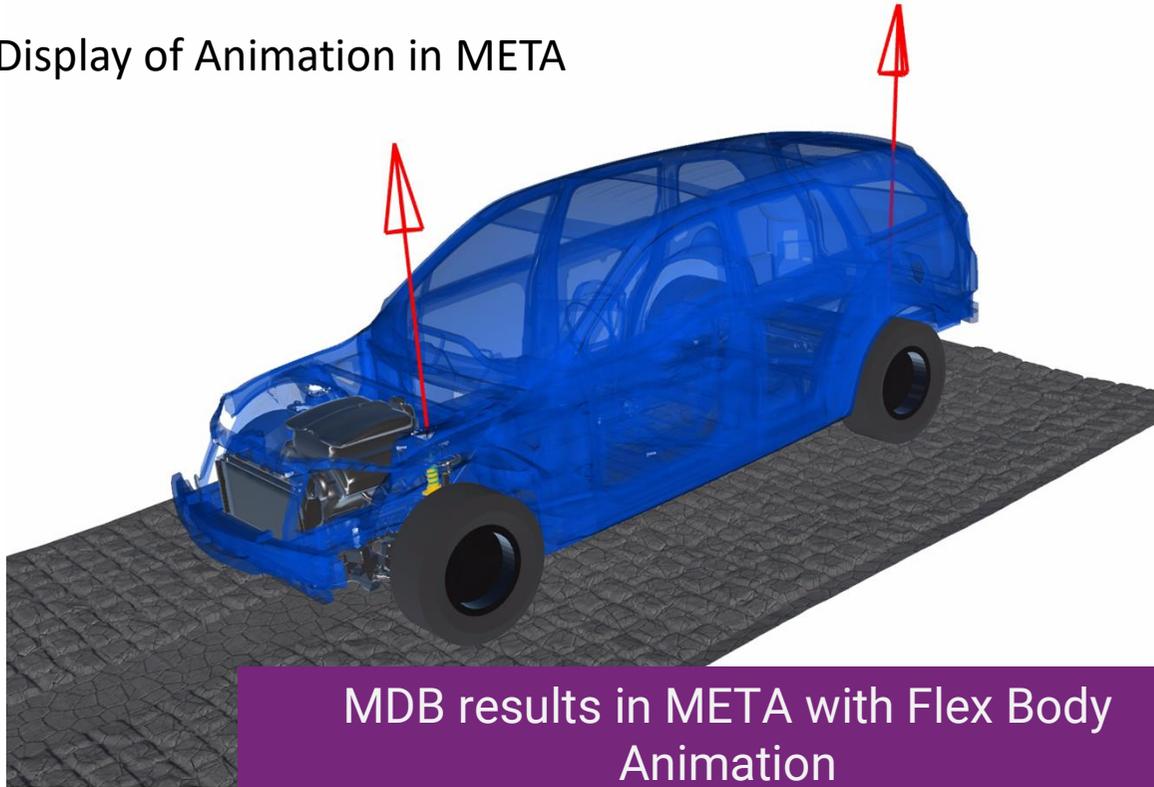


Variants:

- Two different tire formulations:
 - PAC2002 (Pacejka)
 - FTire
- Two different road surfaces modelled:
 - Sine Road (26Hz)
 - Belgian Block Road
- Road Builder supports the parametric construction of different road surfaces

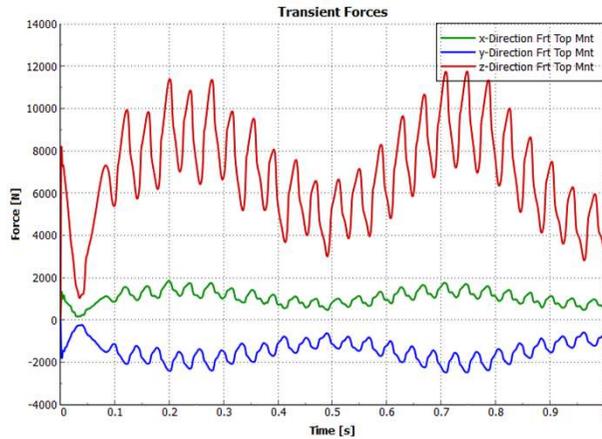
Step 3: Kinetics – Simulation of Driving Car on Road

Display of Animation in META



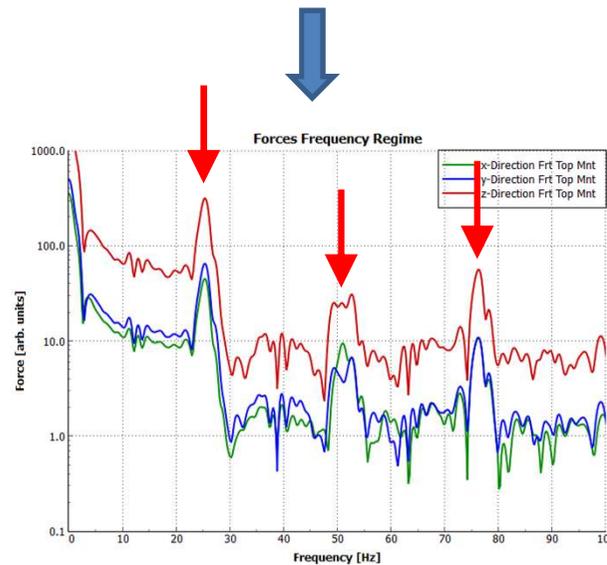
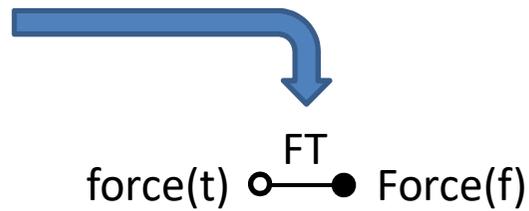
- Several MBD load cases have been simulated in time domain depending on
 - Type of TB: Rigid Body/Flex Body
 - Tire formulation
 - Road surface
- Overall simulation time:
 $T = 1\text{s}$
- Vehicle speed:
 $v = 54\text{Km/h}$

Step 3: Kinetics –Preparation of Loads



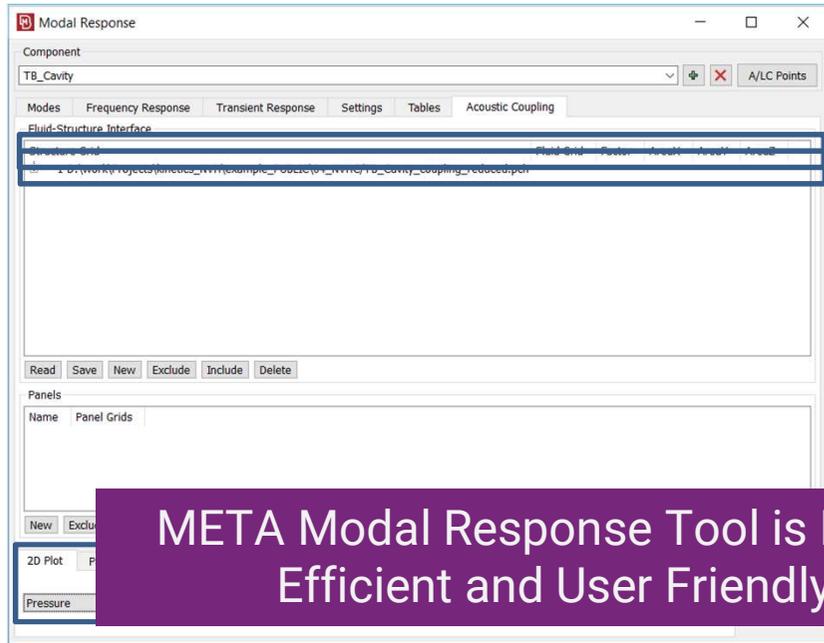
Fourier Transformation (FT):

- $FT \approx t_s \cdot FFT$
- Hanning window
- Zero-padding to increase df



- Python scripting in ANSA and META converts the calculated loads from MBS to Nastran loads
- NumPy package in BETA installation included
- Example: Frt Top Mnt forces for Sine Road
- Higher harmonics due to non-linearities?

Step 4: NVH Post Processing – Modal Response Tool

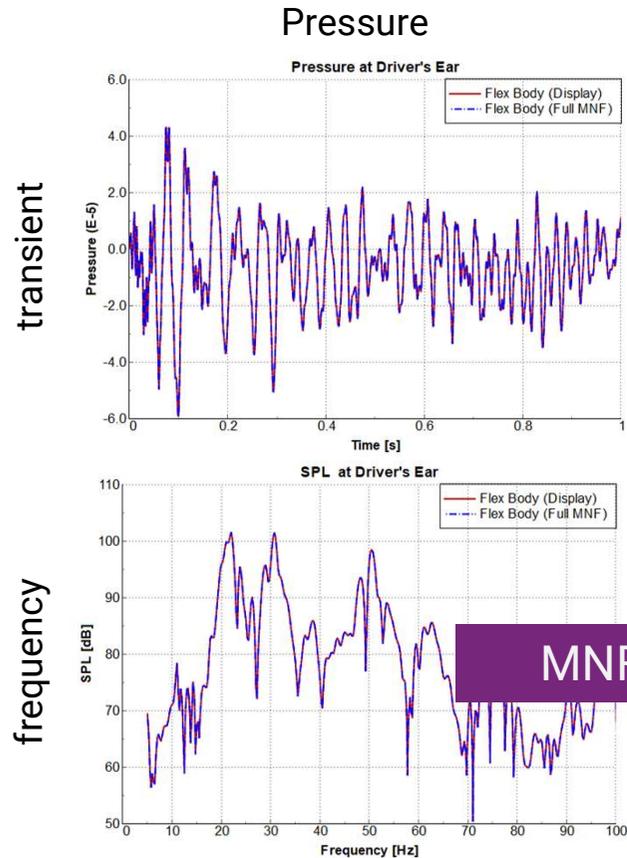


- BUT, transient acoustic response has not been implemented in Modal Response tool yet. It will be implemented in the near future!!
- Transient analysis for acoustic response shown here has been done with Nastran SOL112



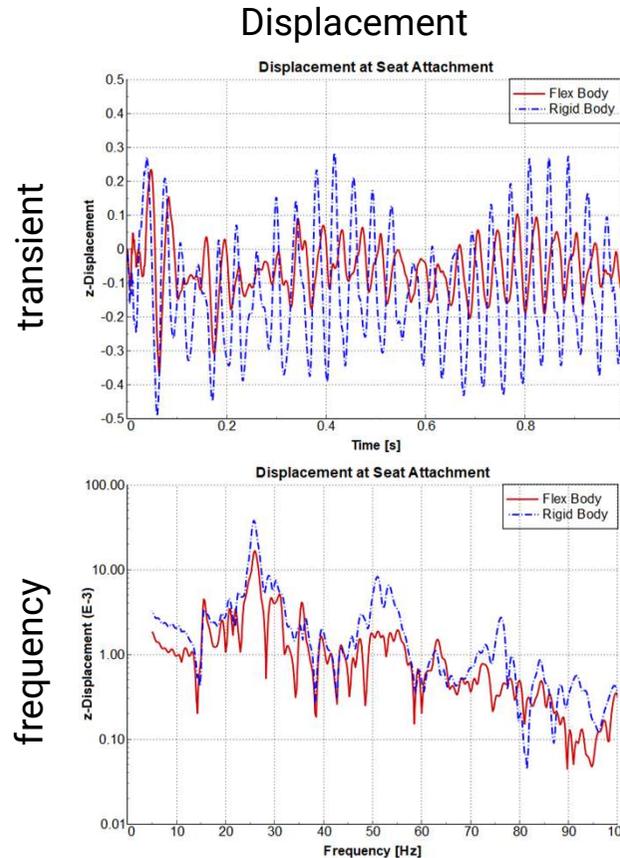
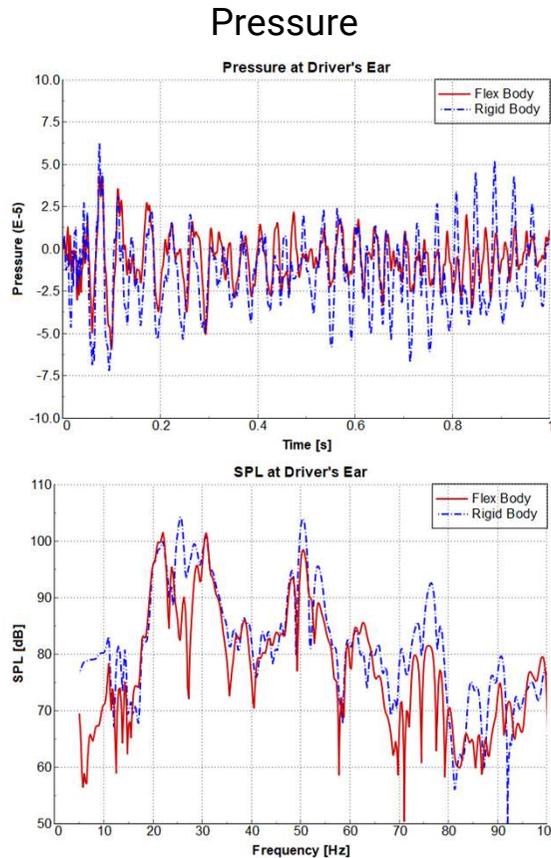
- **Modal Response, Powerful NVH tool in META**
 - 2D/3D transient and frequency response
 - Participation factors
 - Acoustic grid point participation
- **Input:**
 1. Modal result file
 2. FS-Interaction file
 3. Load case file
 4. Freq/time steps
 5. Response Dof
 6. Response type

Step 4: Results – MNF with Display



- Sine Road (26Hz)
- Transient and frequency results
 - pressure at microphone at Driver's Ear position (acoustic)
- Observation:
 - Full MNF = MNF with Display Model

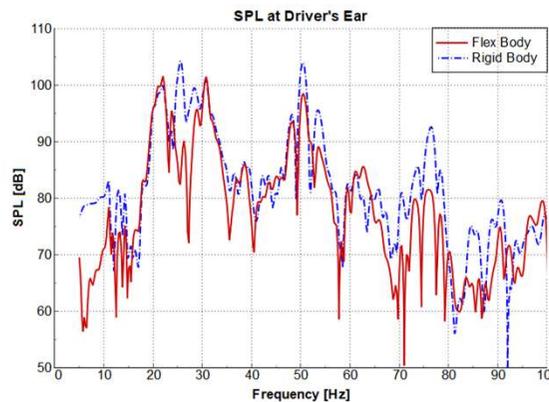
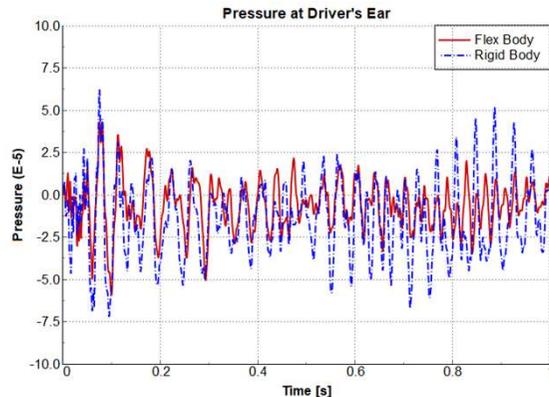
Step 4: Results – Influence of Flex Body



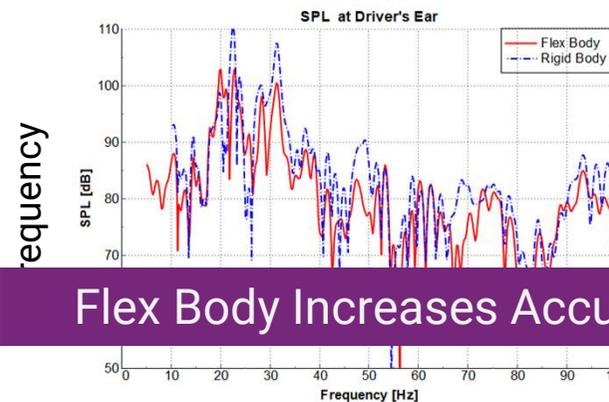
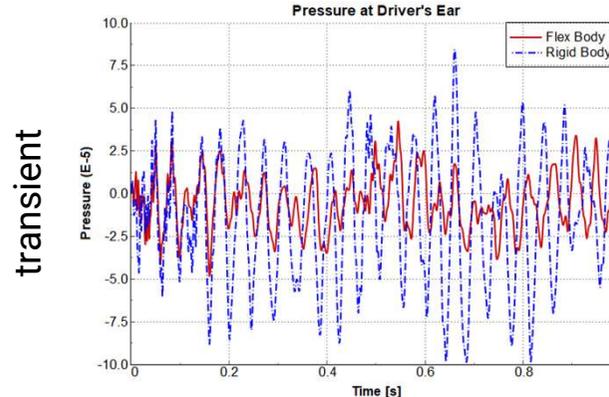
- Sine Road (26Hz)
- Transient and frequency results
 - Pressure at microphone at driver's ear position (acoustic)
 - z-Displacement at seat attachment (tactile)
- **Observation:**
 - Rigid Body leads to increase responses

Step 4: Results – Influence of Flex Body

Sine Road (26Hz)



Belgian Block

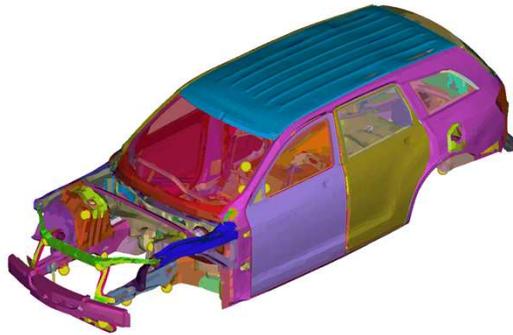


Flex Body Increases Accuracy of Results

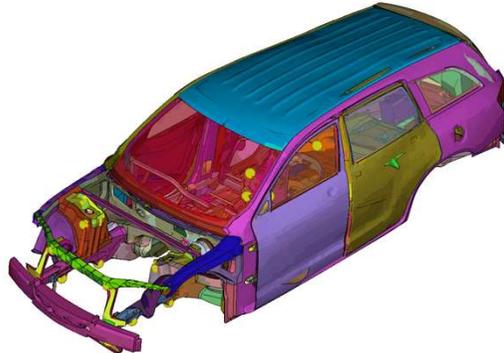


- Sine Road (26Hz) and Belgian Block Road
- Transient and frequency results
 - Pressure at microphone at driver's ear position (acoustic)
- Observation:
 - Rigid Body leads to increase responses

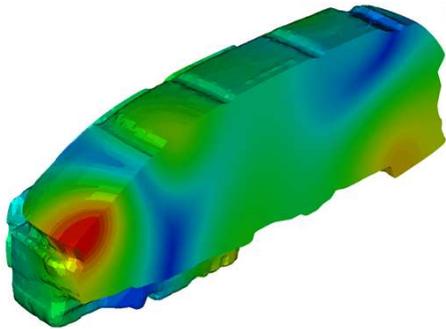
Step 4: Modal Response – 3D Animations



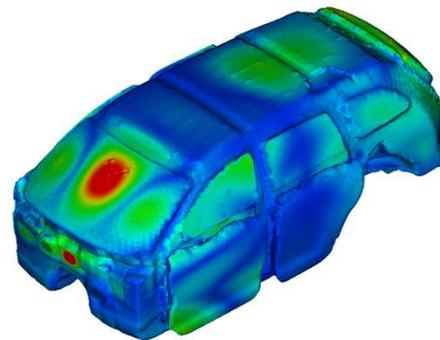
Transient Structural Response



Frequency Structural Response



Frequency Acoustic Response

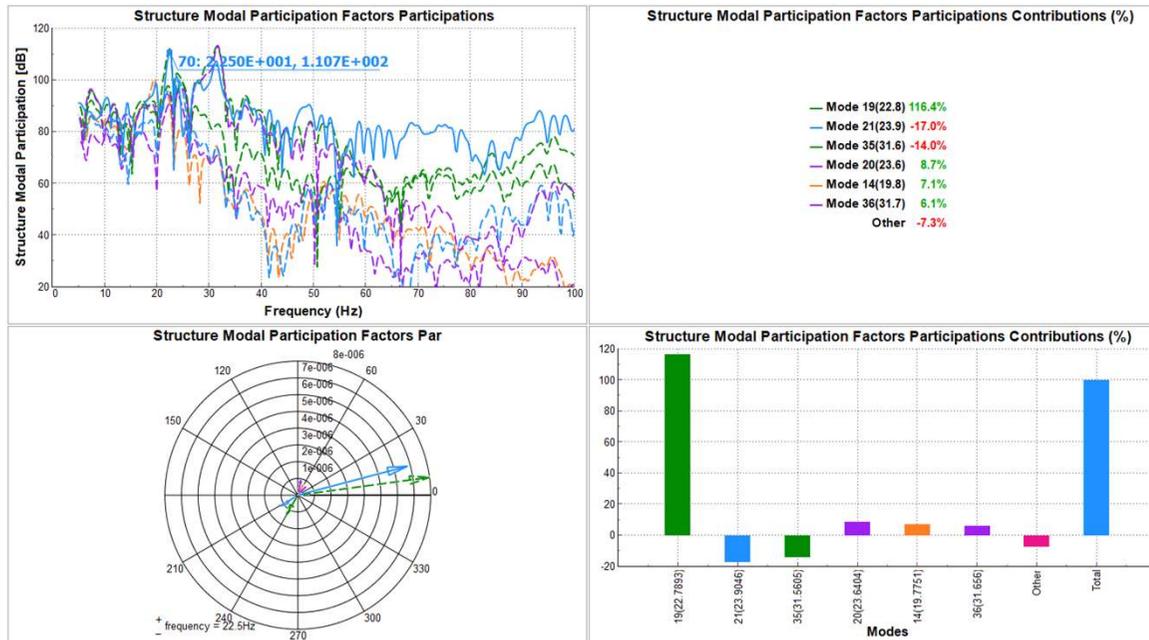


Frequency Grid Point Participation



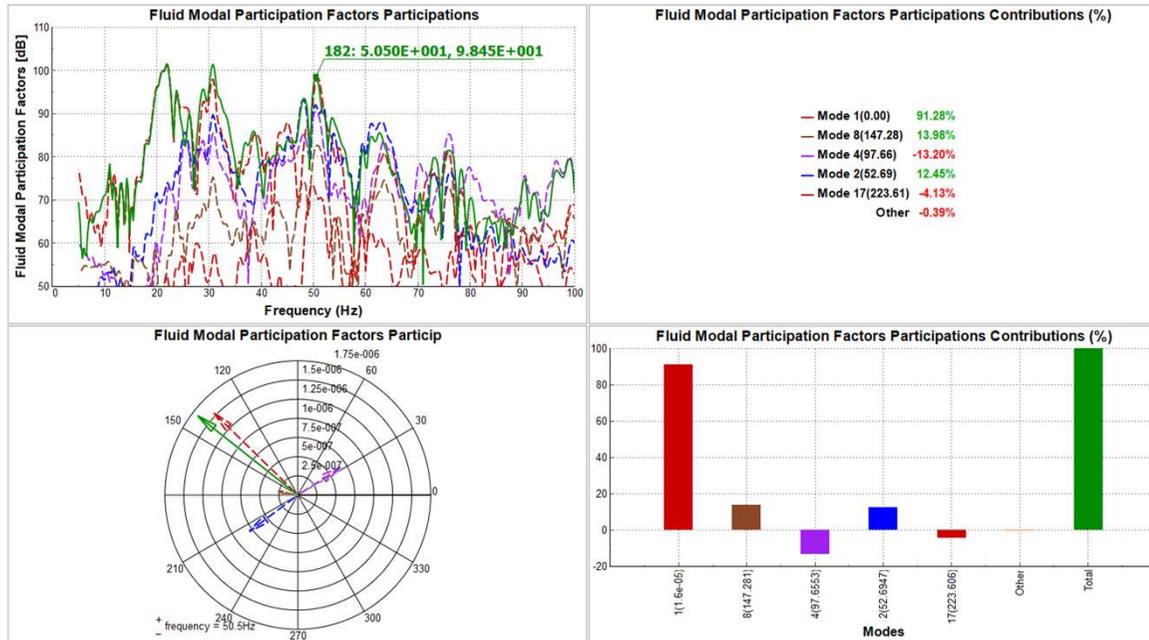
- 3D Animations in Modal Response Tool

Step 4: Modal Response – Structural Modal Participation



- Example:
 - Belgian Block Road, MBD Rigid Body
 - Structural modal participation
 - Main contribution to 22.5Hz peak in SPL: Global Torsion

Step 4: Modal Response – Fluid Modal Participation

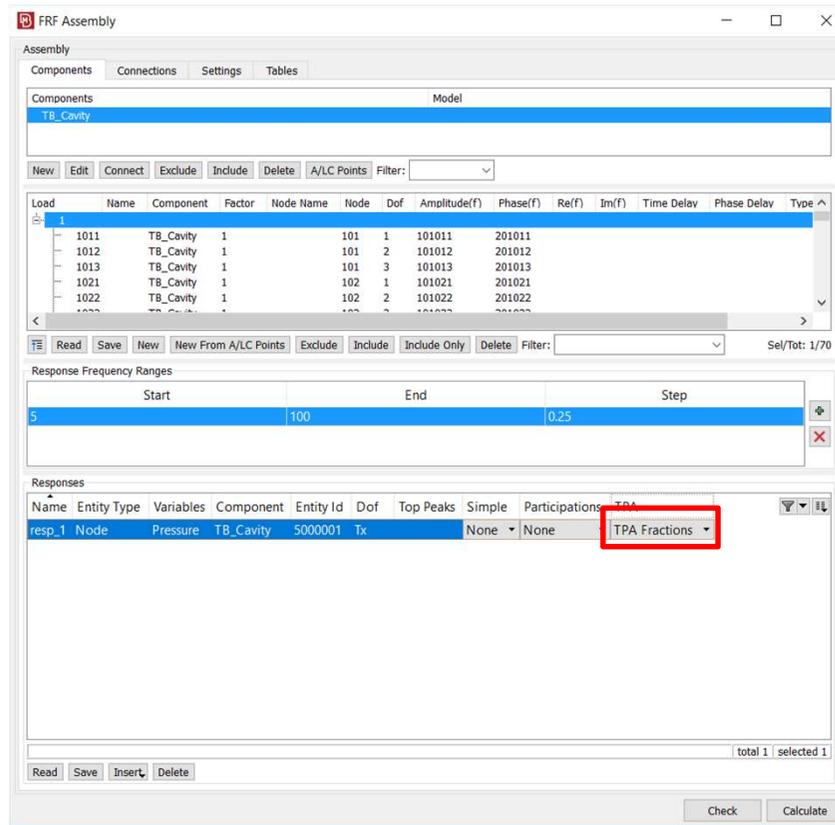


Easy Root Cause Analysis with Modal Participation



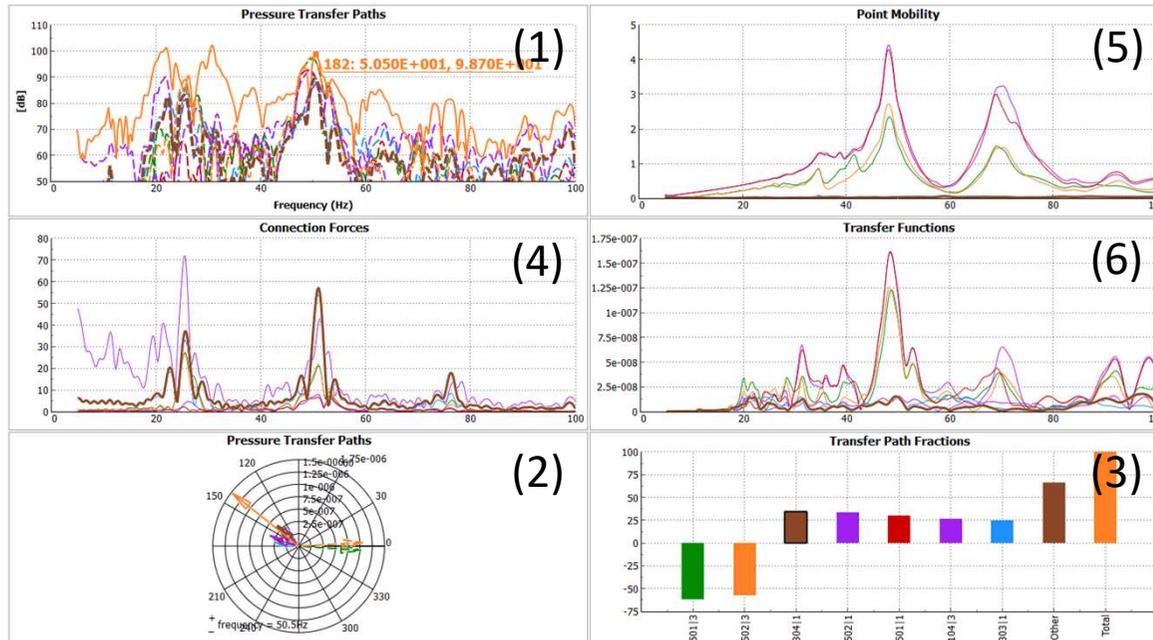
- Example:
 - Sine Road (26Hz), MBD Flex Body
 - Fluid modal participation
 - Main Contribution to 50Hz peak in SPL: Fluid rigid body mode

Step 4: FRF Assembly Tool - Transfer Path Analysis (TPA)



- Advanced NVH analysis in META:
 - TPA in FRF Assembly Tool for any frequency response
 - Reveals which interface Dof and which force component causes mainly the response of the system

Step 4: Results – Transfer Path Analysis



Detailed Root Cause Analysis with TPA



- Example:
 - Sine road (26Hz), MBD Flex Body
 - 50Hz peak in SPL
- TPA Tool displays for N top contributing paths:
 - Path participation
 - Absolute (1)
 - Complex (2)
 - Fraction (3)
 - Forces (4)
 - Point Mobility (5)
 - Transfer functions (6)

Conclusions/Summary

Summary/Results

- A seamless process, completely inside the BETA Suite, was presented using forces from a Non-Linear MBD analysis to calculate NVH related responses on a Trimmed Body
- Using forces calculated with Flex Bodies instead of Rigid Bodies in the MBD simulation has a significant effect on the NVH results
- Utilizing a Display Model during the NVH analysis AND for the MNF file calculation leads to feasible file sizes and especially facilitates the application of Flex Bodies in the MBD simulation
- META NVH tools enable efficient root cause analysis methods

Identified Areas for Tool Improvements

ANSA Kinetics

- Automatic creation of transient and frequency load case (including all necessary options for FT) to avoid scripting solution

META

- Calculation of transient acoustic response
- Modal participation for transient response

Implementation Depends on Customer's
Request/Interests in this Process!

Outlook - Advanced Processes/Applications

Advanced Models

- MBD model with more than one Flex Body, e.g. Subframes also as Flex Bodies
- Linear NVH model with more than one component, e.g. TB and IP separated
 - Coupling of components with FRF assembly or CMS in Epilysis
 - Easy set-up in NVH-Console

Automated Process

- Scripting capabilities in ANSA/META enable automatization and set-up of optimization loops

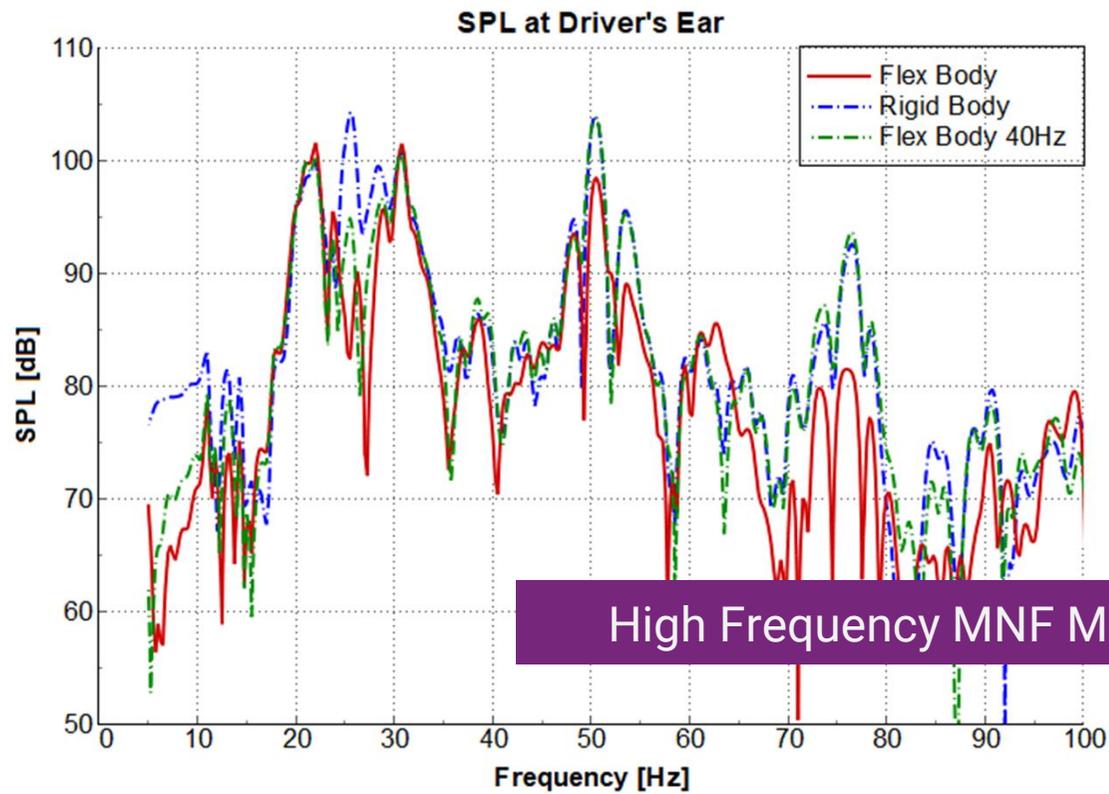


Stay connected

BETA^B
SIMULATION SOLUTIONS

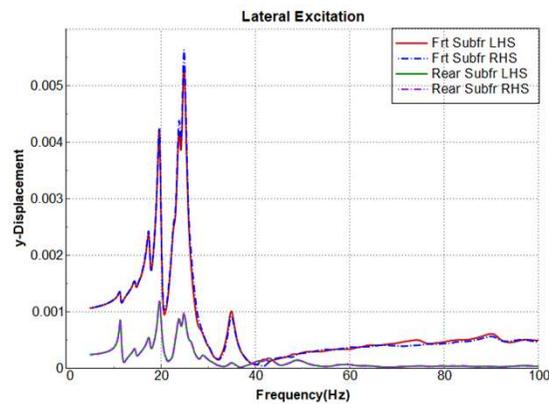
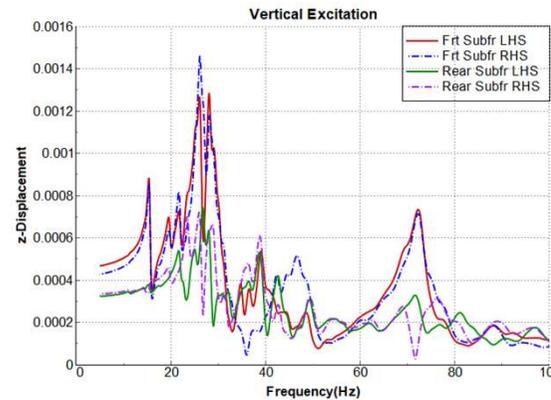
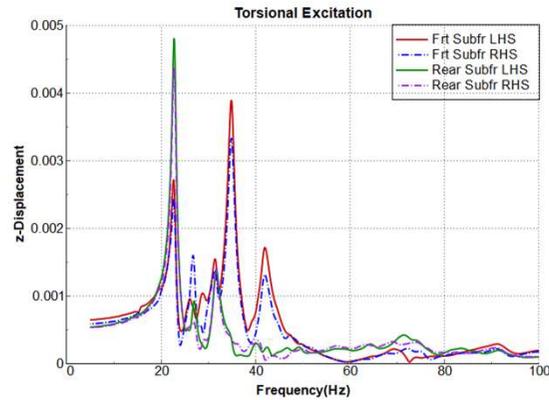
Backup Material

Results – Influence of Constrained Modes



- Example:
 - SPL for sine road

Results – Identification of Global Modes

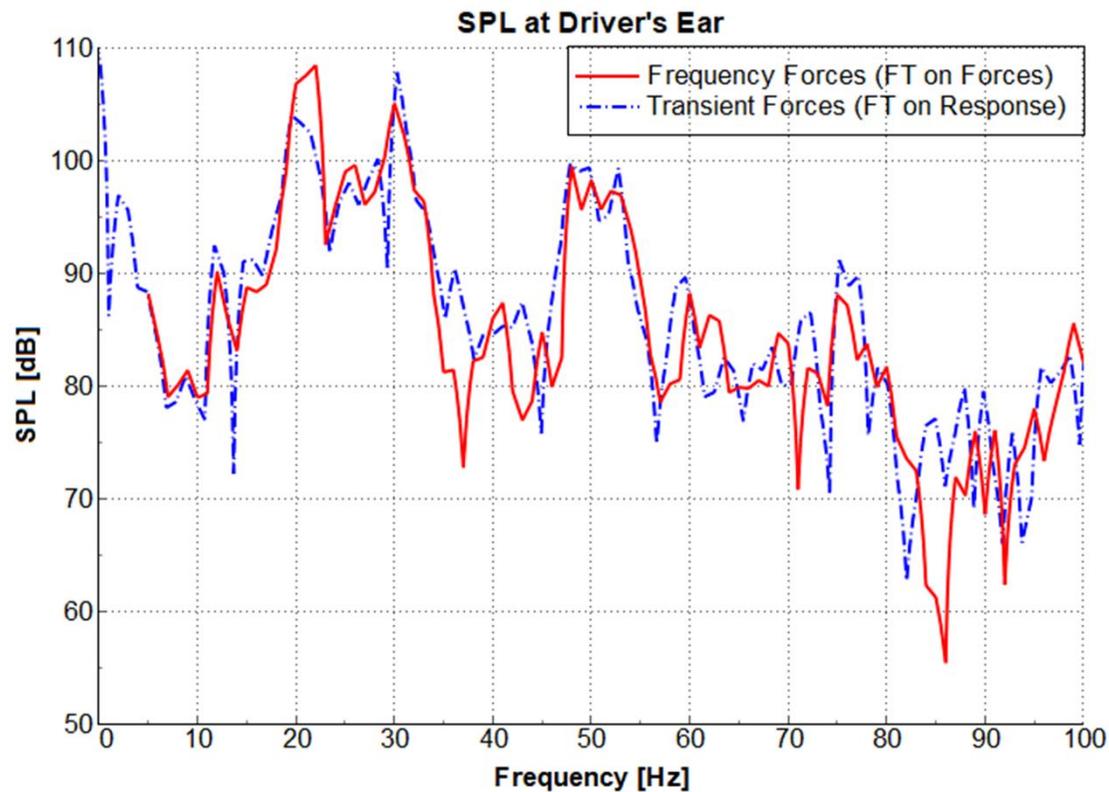


	1. Mode [Hz] (no)	2. Mode [Hz] (no)	3. Mode [Hz] (no)
torsion	22.8 (19)	34.9 (39)	41.7 (54)
lateral	19.6 (15)	24.9 (23)	34.9 (39)
vertical	26.1 (26/27)	28.2 (28)	72.2 (135)



- Global modes are identified by torsional, lateral and vertical dynamic loads
- Main peaks reveal the global modes frequencies
- Modal participation analysis reveals the global mode numbers

Results – Commutability of System Response and FT



- FT on transient response is similar but not identical as frequency response due to FT of forces!
- Possible root cause: Different damping mechanism in frequency and time domain